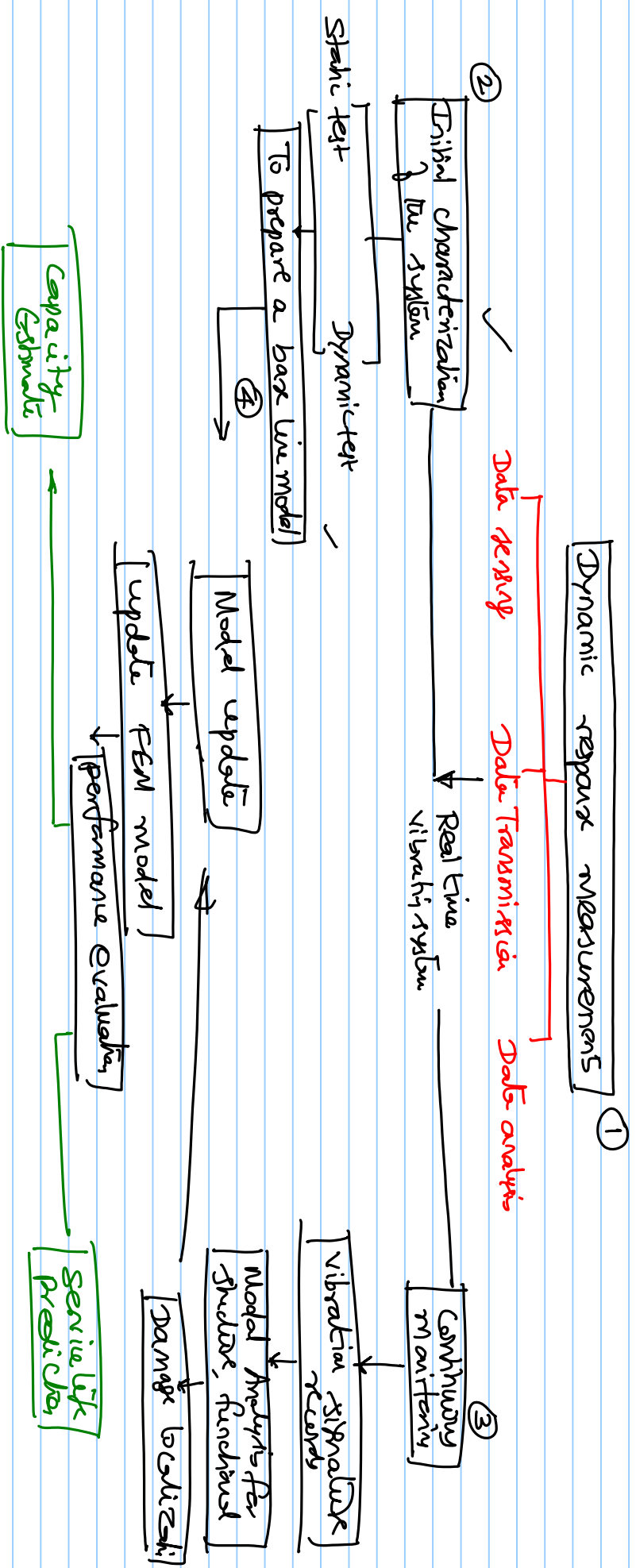


Module 2

Lecture 6 : STM method - I

Flow chart - vibration-based monitoring



I. Method with frequency & mode shapes

- change is structural characteristics can be readily identified by noticing the ^{change is} natural frequencies
- change is $[k]$, $[m]$, change eigenvalue, which can be modelled as:

$$\{z\} = [F]\{\alpha\} + [G]\{k\} \quad (1)$$

where $\{z\}$ = vector of measured frequency changes
 $\{\alpha\}, \{k\}$ vectors of the change in the system \swarrow stiffness respectively
 $[F], [G]$ flexibility, mobility

To calculate the changes in $[k]$ & $[m]$ is to compute $\{\alpha\}, \{k\}$ one needs to calculate the sensitivity matrices.

Eigen λ calculated theoretically from the eigenvalues of the st system (10)

They can be computed numerically using 'perturbation method' with finite element model

— studies conducted earlier show that change is Max matrix, before and after damage is negligible

— Therefore, stiffness matrix, can be reformulated as below:

$$Z_i = \sum_{j=1}^{N_E} F_{ij} \alpha_j \quad \text{--- (2)}$$

where Z_i = fractional change of its eigenvalue (frequency)

α_j = fractional reduction in its stiffness parameter

F_{ij} is expressed as fraction of Modal strain energy for its mode, and is its element's strain

F_{ij} can be expressed as below:

$$F_{ij} = \frac{\{\phi_i\}^T [k_j] \{\phi_i\}}{[\phi_i]^T [k] \{\phi_i\}} \quad (3)$$

where $[k]$ & $[k_j]$ are global & elemental stiffness matrices, respectively.

Once, stiffness matrix for complete system and node displacements are known, the F_{ij} , as seen from Eq(3) can be generated numerically.

To obtain the relative damage, sensitivity Eqn can be reformulated as:

$$\frac{Z_m}{Z_n} = \frac{\sum_{j=1}^{N_E} F_{mj} \alpha_j}{\sum_{j=1}^{N_E} F_{nj} \alpha_j} \quad (4)$$

Suppose, only ① element is damaged, then the above Eqn reduces to

$$\frac{Z_m}{Z_n} = \frac{F_{m1}}{F_{n1}} \quad (5)$$

which is unique for the Q_k loads. Damage can be now located

Form index is given by:

$$C_{ij} = \frac{Z_m}{\sum_{k=1}^{NM} Z_k} - \frac{F_{mq}}{\sum_{k=1}^{NM} F_{kq}} \quad \text{--- (6)}$$

$C_{ij} = 0$, which indicates donor i is location.

Danger propensity is given by:

$$\frac{S w_i^2}{w_i^2} = \eta S_{ik} \left(\frac{a_k}{H_i} \right)^2 \quad \text{--- (7)} \quad \checkmark$$

Where $\frac{\partial u_i^1}{\partial \omega_i^1}$ is factorial charge is expenditure is the i^{th} mode

$\left(\frac{\partial \epsilon_k}{\partial H} \right)_i$ is dimensional crack size, which is normalized to the depth of the member (H).

η - shape factor, accounts for geometry of the mass

S_{ik} is sensitivity of k^{th} location is the i^{th} modal strain energy

If there is a factorial charge is given value of the system, which is measured experimentally, then we can easily determine the crack size from

$$\epsilon_p(z).$$

Eg (3) Key a Limitation:

If only one damage is present, Eg (3) can be used to locate the damage

I can't multiple damage locations, it is not applicable

Based upon the change in $[K]$ & $[M]$ one can observe change in Eigen values, before & after damage

- crack/damage location
 - crack size
- etc

II To estimate mass & stiffness of a shear model building from the model test

Let undamped shear building, represented by the eqn below as below:

$$M \ddot{x} + kx = F \quad (8)$$

Characteristic eqn to determine Eigenvalue mode shape is given by:

$$(k - \omega_i^2 m) \phi_i = 0.$$

where ω_i - Eigen value

ϕ_i - corresponds mode shape

Mass matrix is given by

$$[M] = \begin{bmatrix} m_1 & - & - & - & - \\ 0 & m_2 & - & - & - \\ 0 & - & - & - & - \\ & - & - & - & - \\ & & - & - & m_n \end{bmatrix}$$

Expanding the above equations for L & r modes and reorganizing in terms of stiffness and mass parameters, follows Eqn's presented:

$$\begin{bmatrix}
 \phi_{L1} & -\omega_L \phi_{L1} & \phi_{L1} - \phi_{L2} & \dots & \dots \\
 \phi_{Lr} & -\omega_r \phi_{Lr} & \phi_{Lr} - \phi_{Lr+1} & \dots & \dots \\
 \vdots & \vdots & \vdots & \ddots & \vdots \\
 \vdots & \vdots & \vdots & \ddots & \vdots \\
 \vdots & \vdots & \vdots & \ddots & \vdots
 \end{bmatrix}
 \begin{Bmatrix}
 \phi_{L1} - \phi_{L2} \\
 \phi_{Lr} - \phi_{Lr+1} \\
 \omega_L \phi_{L1} \\
 \omega_r \phi_{Lr} \\
 \vdots
 \end{Bmatrix}
 = \begin{bmatrix} 0 \end{bmatrix}$$

(or) simply

$$[B] \{b\} = \{0\}$$

If $m_n = 1$, then the above equation will reduce to the form as below.

$$\underline{[B']} \{b'\} = \begin{Bmatrix} 0 \\ \vdots \\ \omega_k \phi_{k_n} \\ \omega_{k+1} \phi_{k+1} \end{Bmatrix}$$

where $[B']$ is of the order $(2n-2) \times (2n-2)$ in which the last 2 rows & columns of $[B]$ are eliminated

$\{b'\}$ is $(2n-2) \times (1)$ vector is which last 2 members of $\{b\}$ are eliminated

Now, solving for the unknown mass & stiffness parameters (k_i , m_i) we get:

$$\{b'\} = ([B']^T [B']^{-1}) [B']^T \begin{Bmatrix} 0 \\ \vdots \\ \omega_k \phi_{kn} \\ \omega_n \phi_{nn} \end{Bmatrix}$$

$$k_n = \frac{\omega_k \phi_{kn}}{\phi_{kn} - \phi_{n-1}}$$

Mass and stiffness parameters, obtained from the above set of equations at relative values of m_n ($\therefore m_n$ is considered to be unity)

Advantages of this method

- (1) Only mode shape & frequency
- (2) modes are required
- (2) This can be applicable only to shear model buildings
- (3) This is valid only for undamped systems

Summary

- STM methods
- flow chart - for vibration-based monitoring
 - i) using frequency/mode shapes, how can abnormalities be detected?
 - ii) how to obtain [M] & [K] for shear-model build; with and without frequencies/mode shapes.

