

## Module 2

Lect 7: Damage identification using lumped mass.

Change in eigenvalues - can enable damage detection

Let us consider dynamic eqn of forces acting @ the  $j$ 's story.

By considering all the stories above  $j$ 's story, we get:

$$K_{ij}(j) \Delta u(j) + C_{ij} \dot{\Delta u}(j) = f_{ij}(j) \quad (1)$$

where  $k_{ij}$  and  $c_{ij}$  are stiffness and damping parameters of the  $j^{\text{th}}$  story

$f_{ij}(t)$  is the inertia force acting on the story mass of  $j^{\text{th}}$  story and all the story above it.

$$f_{ij}(t) = - \sum_{k=j}^N m_k \ddot{x}_{ik}(t) \quad (2)$$

$$= - \omega_i^2 \sum_{k=j}^N m_k \phi_{ik}(t) \quad (3)$$

where  $N$  - # of story       $\phi_{ik}(t)$  - is relative displacement  
 $m_k$  - mass of the  $k^{\text{th}}$  story      plus of  $(i-1)^{\text{th}}$  story

for its mode of vibration, this can be expressed as

$$Q_{ij}(t) = x_{ij}(t) - x_{i(i-1)}(t)$$

$$= \left\{ \phi_{ij} - \phi_{i(i-1)} \right\} e^{i\omega_i t} \quad (4)$$

$$\dot{Q}_{ij}(t) = \dot{x}_{ij}(t) - \dot{x}_{i(i-1)}(t)$$

$$= \left\{ \dot{\phi}_{ij} - \dot{\phi}_{i(i-1)} \right\} \omega_i e^{i\omega_i t} \quad (5)$$

Where  $\phi_{i,j}$  &  $\psi_i$  are eigenvectors & eigenvalues of its mode

Once Eigenvalues & eigenvectors are measured for its story,

Eg (1) can be formulated for its story (2) each time interval.

Using short data logs, which is approximately equal to  $\psi_i$  of the building.

Eg (1) can be solved.

One can repeat this procedure for other story to determine  $k_i$  &  $m_i$  parameters for each story.

## Solvent points

- (1) This method assumes a known steady state
- (2) only applicable for lumped mass, shear building model.

(3) Method to identify damage using Element Modal stiffness

In a linear, undamped structure, its modal stiffness is given by:

$$K_i = \bar{\Phi}_i^T [K] \bar{\Phi}_i \quad (1)$$

where  $\bar{\Phi}_i$  is the  $i$ th mode shape vector

$[K]$  is the complete stiffness matrix of the entire structure

Contribution of  $j$ 's member to  $i$ 's nodal stiffness is given by:

$$K_{ij} = \bar{F}_i^T [k_j] \bar{F}_i$$

where  $k_j$  -  $j$ 's member contribution to  $[k]$

Gradient of member energy of  $i$ 's node, contributed by  $j$ 's member is called Member sensitivity.

Member sensitivity is given by:

$$F_{ij} = \frac{P_{ij}}{k_i}$$

The above ~~eqn~~ is valid for undamaged structure.

This is modified for a damaged structure. Model sensitivity for a damaged structure is given by:

$$F_{ij}^* = \frac{k_{ij}^*}{k_i^*}$$

$$k_{ij}^* = \bar{\Phi}_i^{*T} [k_{ij}^*] \bar{\Phi}_i^*$$

$$k_i^* = \bar{\Phi}_i^{*T} [k_i^*] \bar{\Phi}_i^*$$

$$k_j = E_j [k_{j0}]$$

$$k_j^* = E_j^* [k_{j0}]$$



Where  $E_{ij}$ ,  $E_{ij}^*$  represent material stiffness properties, related to undamaged & damaged state of the structure

$[K_{ij}]$  is assumed to be unchanged even after damage occurs.

Basic assumption of the method is that

Modal sensitivity for its mode & its member remains unchanged before & after damage

Mathematically

$$\frac{F_{ij}}{E_{ij}^*} = \frac{k_{ij}^* k_i}{k_i^* k_{ij}} = 1$$

Damage Index,  $\beta_i$  for the  $j^{\text{th}}$  member is defined as:

$$\beta_j = \frac{E_j}{E_j^*}$$

Substituting from the earlier eqn

$$\beta_j = \frac{\gamma_{ij}^* k_i}{\gamma_{ij} k_i^*}$$

$$= \frac{\bar{\phi}_i^{*T} (k_{ij}) \bar{\phi}_i^* k_i}{\bar{\phi}_i^T [k_{ij}] \bar{\phi}_i k_i^*}.$$

Damage Index, can also be approximated as, below.

$$\beta_i \approx \left[ \frac{\bar{\Phi}_i^{*T} [\mathbf{k}_{ij}] \bar{\Phi}_i^* + \sum_{k=1}^{NE} \bar{\Phi}_i^{*T} [\mathbf{k}_{jk}] \bar{\Phi}_i^*}{\bar{\Phi}_i^T [\mathbf{k}_{ij}] \bar{\Phi}_i + \sum_{k=1}^{NE} \bar{\Phi}_i^{*T} [\mathbf{k}_{jk}] \bar{\Phi}_i} \right] \left( \frac{k_i}{k_i^*} \right)$$

Normalized damage indicator is given by:

$$Z_i = \frac{\beta_i - \bar{\beta}}{\sigma_{\beta}} \quad \text{where } \bar{\beta} \text{ mean value of } \beta, \quad \sigma_{\beta} \text{ std deviation of } \beta.$$

Severity of damage can be approxed as:

$$E_j^* = E_j \left( 1 + \frac{\alpha E_j}{E_i} \right)$$
$$= E_j (1 + \alpha_j)$$

$$\text{where } \alpha_j = \frac{\gamma_{ij}^* k_i^*}{\gamma_{ij}^* k_i} - 1$$

(4) Damage identification using Modal Strain Energy

This is a Two stage process:

1<sup>st</sup> stage: to locate the damage using change in modal strain energy of the element

2<sup>nd</sup> stage: Extent of damage is determined by iterative scheme

Modal strain energy of  $j^{\text{th}}$  element of 105 Undamaged & damaged structure is given by:

$$MSE_{ij}^d = \Phi_i^T k_j \Phi_i \quad (1)$$

$$MSE_{ij}^d = \Phi_{di}^T k_{dj} \Phi_{di} \quad (2)$$

where  $d$  subscript identifies damaged state

where  $k_j$  element stiffness matrix of  $j^{\text{th}}$  element  $\Phi_i$  is the  $i^{\text{th}}$  mode shape

Change in Modal strain energy ratio is indicated as  $MSE_{CR}$

$$MSE_{CR,j}^i = \frac{|MSE_{ij}^d - MSE_{ij}|}{MSE_{ij}} \quad (3)$$

E<sub>3</sub> is a meaningful indicator of damaged elements

Damaged elements will have a significant change in the stiffness.   
 - stiffness will be degraded

Change in stiffness for the damaged element can be expressed as a diagonal change to element stiffness matrix.

$$K^d = K + \sum_{j=1}^L \Delta k_j \quad (4)$$

$$= K + \sum_{j=1}^L \alpha_j k_j \quad \text{--- (5)}$$

(valid for  $-1 < \alpha_j \leq 0$ )

Now, change in Model strain energy is expressed as below:

$$MSE_{ij} = 2A \bar{\Phi}_i^T k_j \bar{\Phi}_i + \alpha_j \bar{\Phi}_i^T k_j \bar{\Phi}_i \quad \text{--- (6)}$$

In the above eqn,  $\alpha_j$  is unknown.

To start with, this is assumed to be zero & iteration is set in.

Thus,

$$MSE_{ij} = 2A \bar{\Phi}_i^T k_j \bar{\Phi}_i \quad \text{--- (7)}$$

Model strain energy change can be determined for both damaged & undamaged spots. By using the appropriate ( $k_j$  &  $\bar{\Phi}_i$ ) - for damaged case, one can use  $k_{dj}$ ,  $\bar{\Phi}_{id}$ .



In the undamped case, following Eqn holds good:

$$\left[ (K + \Delta K) - (\omega_i + \Delta\omega_i) M \right] [\bar{x}_i + \Delta\bar{x}_i] = 0. \quad (8)$$

Now,  $\Delta\bar{x}_i$  is expressed as a linear combination of mode shape of undamped system & is given by:

$$\Delta\bar{x}_i = \sum_{k=1}^n c_{ik} \bar{x}_k \quad (9)$$

Sub (9) in Eq (8) & neglecting higher order terms, we get

$$c_{ik} = - \frac{\bar{x}_r^T \Delta K \bar{x}_i}{(\omega_r - \omega_i)} \quad \text{for } r \neq i \quad (10)$$

MSEC is then given by:

$$MSEC_{ij} = 2 \Phi^T k_j \left( \sum_{r=1}^n - \frac{\Phi_r^T \Delta k \Phi_i}{(\omega_r - \omega_i)} \right) \Phi_r \quad \text{for } r \neq i \quad (11)$$

The earlier Eqn (4-5) of MSEC can be simplified as below:

$$MSEC_{ij} = \sum_{j=1}^L 2 \alpha_p \Phi_i^T k_j \left( \sum_{r=1}^n - \frac{\Phi_r^T k_p \Phi_i}{\omega_r - \omega_i} \right) \Phi_r \quad \text{for } r \neq i \quad (12)$$

Once the damage is located, then damage severity can be determined as below:

$$\begin{Bmatrix} MSEC_{i1} \\ MSEC_{i2} \\ \vdots \\ MSEC_{ij} \end{Bmatrix} = \begin{bmatrix} \beta_{11} & \beta_{12} & \dots & \beta_{1p} \\ \beta_{21} & & & \\ \vdots & & & \\ \beta_{j1} & & & \beta_{jp} \end{bmatrix} \begin{Bmatrix} \alpha_1 \\ \alpha_2 \\ \vdots \\ \alpha_p \end{Bmatrix} \quad (4.3)$$

where  $\beta$  is a suspected damaged site (location)  
 $j$  is elements considered to compute  $MSEC$

$$\beta_{jk} = - \frac{2 \sum_{r=1}^n \Phi_i^T k_r \frac{\Phi_r^T k_i \Phi_i}{(\omega_r - \omega_i)}}{\Phi_i^T} \quad \text{for } r \neq i$$

where  $\beta_{jk}$  is the element  $j$ th row  $k$ th column of  $MSE_C$

$MSE_C$  are experimentally measured and the substituted in Eq (13). As obtain the fractional change in stiffness the damped system (elements)

once, initial value of  $\Delta p$  (as referred is  $\Delta p_0$ ) is obtained,

values of  $MSE_C$  can be updated for each iteration, until convergence is reached

## Summary

- Damage identification via lumped mass
- " " element model stiffness
- " " modal strain energy method





