

# Smart Materials, Adaptive Structures, and Intelligent Mechanical Systems

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# Lecture 27

## Simply Supported Plates with Normal Load

# References for this Lecture

1. Analysis and Performance of Fiber Composites, Agarwal, B.D. and Broutman, L. J., John Wiley & Sons.
2. Mechanics of Composite Materials, Jones, R. M., Mc-Graw Hill
3. Structural Analysis of Laminated Composites, Whitney, J. M., Technomic
4. Nonlinear Analysis of Plates, Chia, C., McGraw-Hill International Book Company

# Lecture Overview

- Analysis of a Simply Supported Plate on All Four Sides
  - Governing equations
  - Boundary conditions
  - Stiffness matrices
  - Series solution
- Assessing Convergence of Solution

# Simply-Supported Plate on All Sides

- Consider a plate of dimensions  $a \times b$  in  $x$  and  $y$  directions, respectively, and which is simply supported on all four sides.
- Further, we assume that this plate has a symmetric and specially orthotropic lamination sequence. Thus,
  - $[B] = [0]$  due to symmetry
  - $A_{16} = A_{26} = D_{16} = D_{26} = 0$
- Finally we assume that the plate is normally loaded as shown in Fig. 27.1 with a load intensity of  $q(x,y)$ .

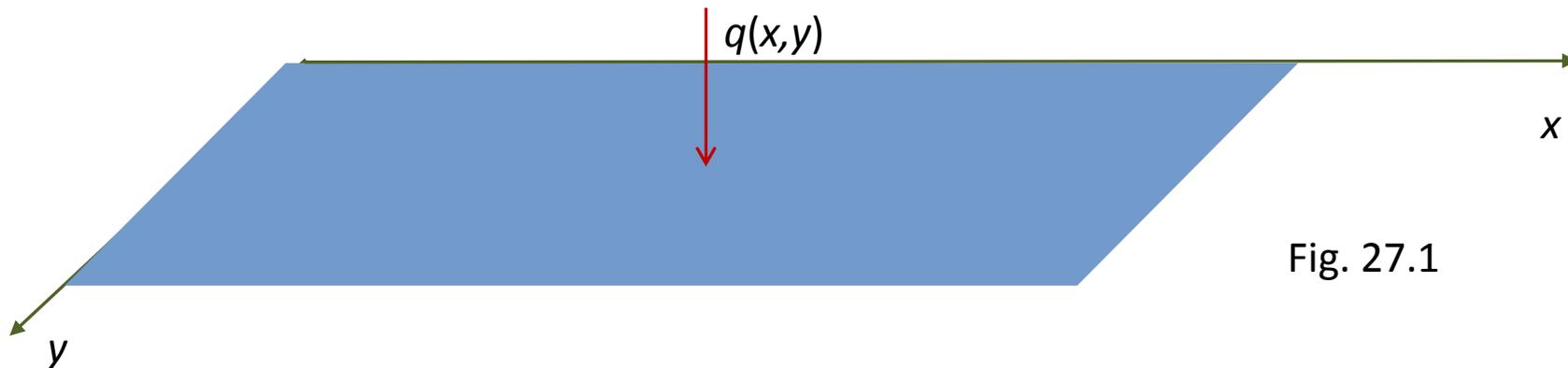


Fig. 27.1

# Simply-Supported Plate on All Sides

- For such a plate, the out-of-plane boundary conditions are:
  - $w^\pm = 0$  on all four sides.
  - $M_x^\pm = 0$  at  $x = 0, a$ .
  - $M_y^\pm = 0$  at  $y = 0, b$ .
- As explained earlier, the governing equation for equilibrium for out-of-plane direction for such a plate is decoupled with in-plane equations, because the plate's lamination sequence is symmetric. This equation is reproduced below.

$$\frac{\partial^2 M_x}{\partial x^2} + 2 \frac{\partial^2 M_{xy}}{\partial x \partial y} + \frac{\partial^2 M_y}{\partial y^2} + q = 0$$

- Expressing this equation in terms of derivatives of  $w^0$ , we get:

$$q(x, y) = D_{11} \frac{\partial^4 w^0}{\partial x^4} + 2(D_{12} + 2D_{66}) \frac{\partial^4 w^0}{\partial x^2 \partial y^2} + D_{22} \frac{\partial^4 w^0}{\partial y^4} \quad (\text{Eq. 27.1})$$

# Simply-Supported Plate on All Sides

- Equation 27.1 has to be solved for  $w^0$ . At this stage, we assume a series function for  $w^0$ , which satisfies the boundary conditions listed earlier. Let this assumed function be:

$$w^0(x, y) = \sum \sum w_{mn} \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b}$$

where,  $m$ , and  $n$  are integers which can vary between 1 and infinity.

- Such a series expression for  $w^0$  satisfies out-of-plane boundary conditions. If this expression also satisfies Eq. 27.1, then it is a valid solution. For this expression to satisfy Eq. 27.1, we put it into the equation, and get the following condition, which has to be satisfied for getting a valid solution for  $w^0$ .

$$q^0(x, y) = \sum \sum \left\{ D_{11} \left( \frac{m\pi}{a} \right)^4 + 2(D_{12} + 2D_{66}) \left( \frac{m\pi}{a} \frac{n\pi}{b} \right)^2 + D_{22} \left( \frac{n\pi}{b} \right)^4 \right\} w_{mn} \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b}$$

(Eq. 27.2)

# Simply-Supported Plate on All Sides

- Now, if  $q(x,y)$  can also be expressed as a double Fourier series, as shown below:

$$q(x,y) = \sum \sum q_{mn} \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b}$$

then, for Eq. 22.2 to be true, the following condition must hold.

$$w_{mn} = \frac{q_{mn}}{d_{mn}}$$

where,

$$d_{mn} = D_{11} \left( \frac{m\pi}{a} \right)^4 + 2(D_{12} + 2D_{66}) \left( \frac{m\pi}{a} \frac{n\pi}{b} \right)^2 + D_{22} \left( \frac{n\pi}{b} \right)^4$$

- Hence, if we know  $q_{mn}$ , we can find out-of-plane deflections for such a plate.

# Simply-Supported Plate on All Sides

- The values for  $q_{mn}$  can be gotten by decomposing  $q(x,y)$  into a double Fourier series as shown below.

$$\int_0^b \int_0^a q(x,y) \left[ \sin \frac{i\pi x}{a} \sin \frac{j\pi y}{b} \right] dx dy = \int_0^b \int_0^a q_{mn} \left[ \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} \right] \left[ \sin \frac{i\pi x}{a} \sin \frac{j\pi y}{b} \right] dx dy$$

- While solving for  $q_{mn}$ , following identities may be used.

$$\int_0^a \left[ \sin \frac{m\pi x}{a} \sin \frac{i\pi x}{a} \right] dx = \frac{1}{2} \delta_{im}$$

$$\int_0^b \left[ \sin \frac{n\pi y}{b} \sin \frac{j\pi y}{b} \right] dy = \frac{1}{2} \delta_{jn}$$

- If functions are more complex, then Gaussian integration may be used.
- Also, if there are point loads, then Dirac-Delta functions  $\delta[(x-x_0), (y-y_0)]$  may be used while computing  $q_{mn}$  using above formula.

# Convergence of Solution

- While computing the out-of-plane response for the plate, one has to estimate the number of terms to be used in serial expansion of  $q(x,y)$ . Theoretically, the larger the number of terms the more accurate (or converged) is the solution.
- But then, one question that requires careful consideration relates to the context in which accuracy/convergence is being assessed. Are we looking for convergence of displacement, or stress, or load, or ... ?
- In this context, consider that  $q(x,y)$  is constant over the plate, and has a value  $q_o$ . For such a load, the Fourier series expansion is:

$$q(x,y) = q_o = \sum \sum q_{mn} \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} = \frac{16q_o}{\pi^2} \sum \sum \frac{1}{mn} \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b}$$

where,  $m = 1, 3, 5, \dots$  and  $n = 1, 3, 5, \dots$

# Convergence of Solution

- Here, for a two-term solution (i.e.  $m$ , and  $n$  can assume two distinct values):

$$q_{1,1} = (16q_o/\pi^2) \quad q_{3,3} = (16q_o/\pi^2) \times (1/3^2)$$

- Similarly, for a 20-term solution:

$$q_{1,1} = (16q_o/\pi^2) \quad q_{39,39} = (16q_o/\pi^2) \times (1/39^2)$$

- Hence, we see that load converges in a  $1/m^2$  way.
- Next, we look at convergence of deflection. Earlier, for this plate, it has been shown that:  
$$w_{mn} = q_{mn}/d_{mn}$$
- But, we know that  $d_{mn}$  and  $q_{mn}$  are directly proportional to  $m^4$  and  $1/m^2$ , respectively. Thus, deflection converges in a  $1/m^6$  fashion.

# Convergence of Solution

- Similarly, we can find the pace of convergence for other parameters. The following table lists the rate of convergence for different mechanical parameters as well.

Load	$m^{-2}$
Out-of-plane shear	$m^{-3}$
Curvature	$m^{-4}$
Resultant Moment	$m^{-4}$
Stress	$m^{-4}$
Displacement ( $w$ )	$m^{-6}$

- Hence, we find that among the unknowns, out-of-plane shear converges much more slowly than out-of-plane displacement.