







## Module 5: Premixed Flame

### Lecture 26: Flammability Limits

#### The Lecture Contains:

-  [Flammability Limits](#)
-  [Effect of Pressure on Limit Mixture](#)
-  [Ignition](#)
-  [Flame Stabilization](#)
-  [Flame Stabilization by Burner Rim](#)
-  [Turbulent Premixed Flame](#)

 **Previous**   **Next** 

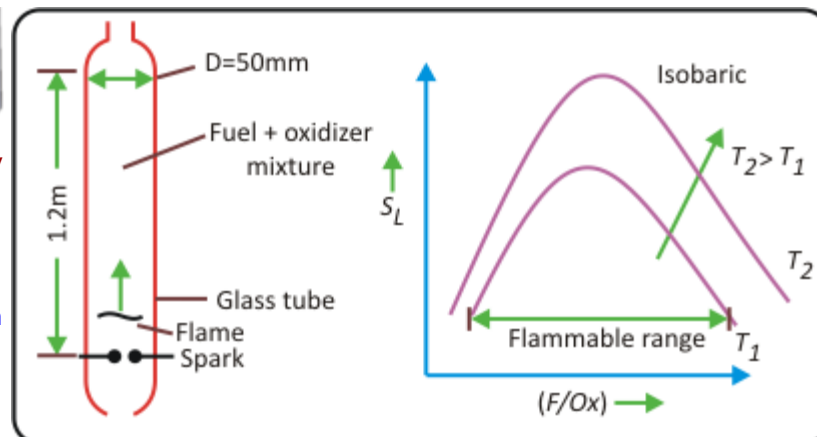
## Flammability Limits

### Limit mixture

Flame can propagate only within a range of fuel-oxidizer ratio

Instrument to determine flammability limit

Vertical glass tube of 1.2 m length and 50 mm ID

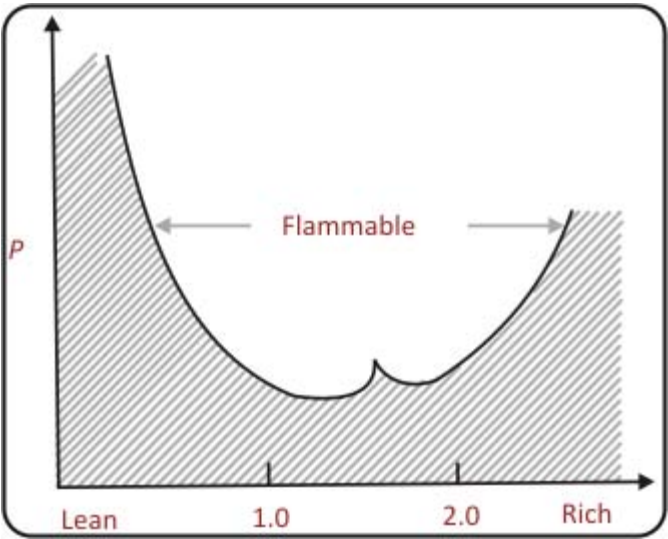


(Figure 26.1)

- Flammability limit is determined by upward propagation of flame from ignition source.
- Mixture is said to be flammable only when it travels above half of the tube.
- Large dia tube is preferred as it provides consistent results and is free from quenching.
- Direction of flame propagation affects the flammability limit.
- Flammability limit varies linearly with temperature.

Effect of Pressure on Limit Mixture

- Pressure has significant effect on flammability limit
- Lean side is not affected significantly by pressure
- Rich limit becomes much wider with increase in pressure



(Figure 26.2)

Fuel	Oxidizer	Stoichiometry (% Fuel)	LFL (%)	UFL (%)
Methane	Air	9.5	5	15
Ethane	Air	5.6	2.8	12.4
Propane	Air	5.6	2.1	9.1
CO	Air	29.5	12	74.2
Hydrogen	Air	29.2	4	74.2

Ignition

Ignition

Rate of heat liberation near the ignition zone  
> rate of heat loss by conduction

Energy generated in flame = Enthalpy × Mass

$$MIE = C_p(T_F - T_u) \left[ \left( \frac{\pi}{4} d_q^2 \right) S_L \rho_u \right]$$

Substituting quenching diameter

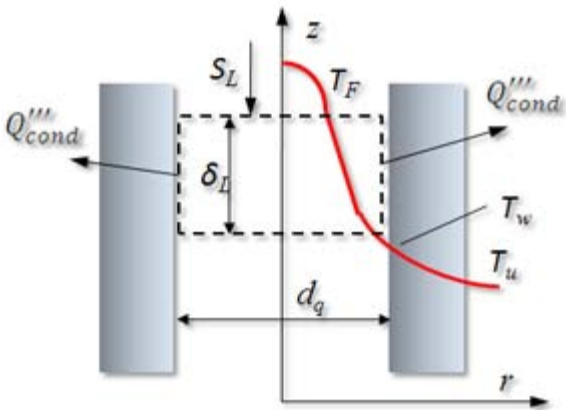
$$MIE = 2\pi C_p \rho_u C_p (T_F - T_u) S_L^3$$

Substituting flame thickness

$$MIE = \frac{128\pi C(T_F - T_u)\alpha^2 k_g}{27 S_L^3}$$

Dependence on pressure

$$MIE \approx \frac{1}{\rho_u^2 S_L^3} \approx P^{-(3n/2-1)}$$

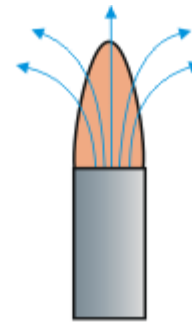


(Figure 26.3)

Fuel-Air	MIE (mJ)
Methane-air	0.47
Ethane-air	0.4
Butane-air	0.34
Acetylene-air	0.03
CO-air	0.05
Hydrogen-air	0.02

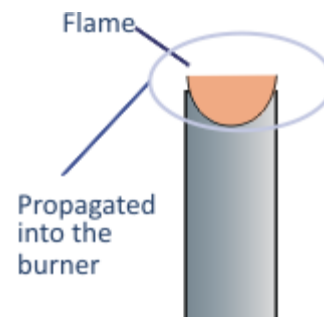
## Flame Stabilization

**Local gas flow velocity** = Local burning velocity



Stream lines through the  
laminar premixed flame

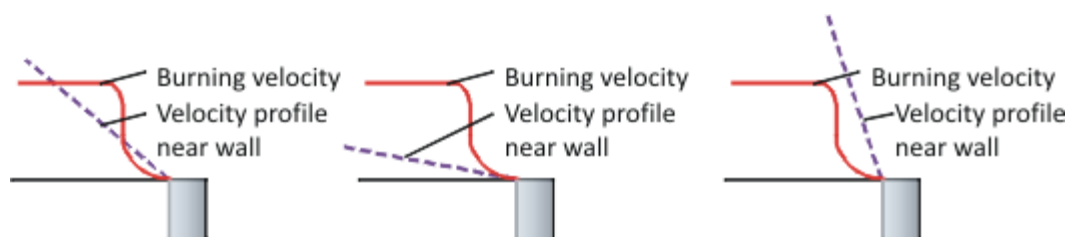
(Figure 26.4)



(Figure 26.5)

◀ Previous    Next ▶

### Flame Stabilization by Burner Rim



(Figure 26.6)

### Stability of flame front near the rim of Bunsen burner

- At burner rim, flow velocity  $\approx$  burning velocity; flame likely to get stabilized here
- Heat loss and radical loss at the burner rim is the cause for flame stabilization
- Laminar velocity profile can be obtained at low Reynolds number
- When flow velocity ( $V_F$ ) < burning velocity ( $S_L$ ),

➤ Flame enters to burner, leading to flash back

➤ At the critical condition, the velocity gradient ( $g_F$ ) is

$$g_F = \lim_{r \rightarrow R} \left( -\frac{dV}{dr} \right)$$

The parabolic velocity profile is  $V = n (R^2 - r^2)$ ;  $R$ -tube radius;

$$n = -\frac{1}{4\mu} (\Delta P / L)$$

$\Delta P$  is pressure difference across tube length,  $\mu$  is the fluid viscosity

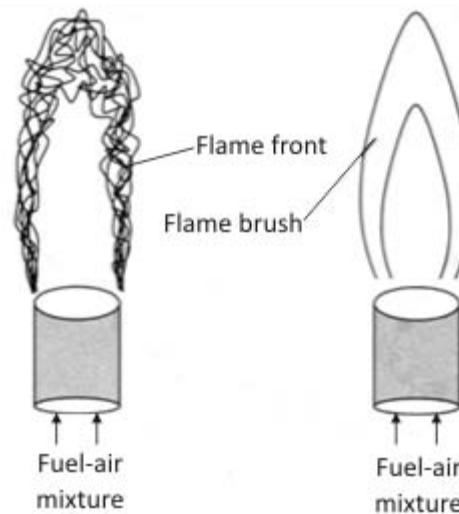
At flash back,  $g_b = 8 \frac{V_{av}}{d}$

## Turbulent Premixed Flame

### Turbulence in flame

- Affects flame propagation rate
- Does not alter chemistry
- Mixing occurs due to random motion of eddies

- Turbulent flames are chaotic in nature
- Instantaneous flame front is highly convoluted
- Actual position of reaction zone moves rapidly in space w.r.t time
- This makes flame to appear thick
- **Turbulent flame brush:** Virtual turbulent flame thickness
- **Laminar flamlets:** Instantaneous reaction zone



(Figure 26.7)

◀ Previous    Next ▶