





Module 5: Premixed Flame

Lecture 23: Structure of 1D Premixed Flame

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Module 5: Premixed Flame

Lecture 23: Structure of 1D Premixed Flame

Laminar Flame Theory

Assumptions:

- 1D, steady, inviscid flow.
- Flame is quite thin.
- Ignition temperature is very close to flame temperature.
- No heat loss including radiation; \Rightarrow Adiabatic flame.
- Pressure difference across the flame is negligibly small.
- Binary diffusion, Fourier and Fick's law are valid.
- Unity Lewis number.
- Constant transport properties ($k_g, C_p, \mu, D \sim \text{constant}$)

Mass conservation: $\frac{d}{dx}(\rho V_x) = 0 \Rightarrow \dot{m}'' = (\rho V_x) = \text{const} \quad \dots (1)$

Species conservation:

Energy equation:

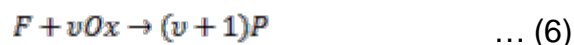
Fuel: $\rho V_x \frac{dY_F}{dx} = \rho D \frac{d^2 Y_F}{dx^2} + \dot{m}_F''' \quad \dots (2)$

$\rho V_x C_p \frac{dT}{dx} = k_g \frac{d^2 T}{dx^2} - \sum \hat{h}_{f,i}^0 \dot{m}_i''' \quad \dots (5)$

Oxidizer: $\rho V_x \frac{dY_{Ox}}{dx} = \rho D \frac{d^2 Y_{Ox}}{dx^2} + \dot{m}_{Ox}''' \quad \dots (3)$

Global reaction mechanism:

Product: $\rho V_x \frac{dY_P}{dx} = \rho D \frac{d^2 Y_P}{dx^2} + \dot{m}_P''' \quad \dots (4)$



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Laminar Flame Theory (Contd.)

Heat release due to chemical reaction:

$$\sum \hat{h}_{f,i}^0 \dot{m}_i''' = [\hat{h}_{f,F}^0 \dot{m}_F''' + v \hat{h}_{f,O_2}^0 \dot{m}_F''' - (v+1) \hat{h}_{f,P}^0 \dot{m}_F'''] = \dot{m}_F''' \Delta \hat{H}_c \quad \dots (7)$$

Now energy equation becomes

$$\dot{m}'' C_p \frac{dT}{dx} = k_g \frac{d^2 T}{dx^2} - \dot{m}_F''' \Delta \hat{H}_c \quad \dots (8)$$

Boundary conditions (For preheat zone)

$$x = -\infty; T = T_u; \frac{dT}{dx} = 0$$

$$x = +\infty; T = T_F; \frac{dT}{dx} = 0$$

Recasted energy equation for preheat zone,

$$\dot{m}'' C_p \frac{dT}{dx} = k_g \frac{d^2 T}{dx^2} \quad \dots (9)$$

Heat transfer due to conduction is balanced by convective heat transfer.

In the reaction zone,

$$x = -\infty; T = T_u; \frac{dT}{dx} = 0$$

$$x = -x_{ig}; T = T_{ig}$$

$$\left. \frac{dT}{dx} \right|_{ig} = \frac{\dot{m}'' C_p}{k_g} (T_{ig} - T_u) \quad \dots (10)$$

$$k_g \frac{d^2 T}{dx^2} = \dot{m}_F''' \Delta H_c \quad \dots (11)$$

Rewriting, E.g. (11).

$$dT \frac{d}{dx} \left(\frac{dT}{dx} \right) = \frac{\Delta H_c}{k_g} \dot{m}_F''' dT \quad \dots (12)$$

$$\left(\frac{dT}{dx} \right)_{ig} = \left[\frac{2 \hat{H}_c}{k_g} \int_{T_{ig}}^{T_F} \dot{m}_F''' dT \right]^{0.5} \quad \dots (13)$$

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Laminar Flame Theory (Contd.)

Combining equations (10) and (13),

$$\frac{\dot{m}'' C_p}{k_g} (T_{ig} - T_u) = \left[\frac{2\Delta\hat{H}_c}{k_g} \int_{T_{ig}}^{T_F} \dot{m}_F''' dT \right]^{0.5} \quad \dots (14)$$

$$\Rightarrow \dot{m}'' = \frac{k_g}{C_p} \frac{1}{(T_{ig} - T_u)} = \left[\frac{2\Delta\hat{H}_c}{k_g} \int_{T_{ig}}^{T_F} \dot{m}_F''' dT \right]^{0.5} \quad \dots (15)$$

Also,

$$\dot{m}'' = \rho_u S_L \quad \dots (16)$$

Combining equations (15) and (16),

$$S_L = \frac{k_g}{\rho_u C_p} \frac{4}{3(T_F - T_u)} \left[\frac{2\Delta H_c}{k_g} \int_{T_{ig}}^{T_F} \dot{m}_F''' dT \right]^{0.5} \quad \dots (17)$$

Mean fuel burning rate per unit volume,

$$\bar{\dot{m}}_F'' = \frac{1}{(T_F - T_u)} \left[\int_{T_u}^{T_F} \dot{m}_F''' dT \right] \quad \dots (18)$$

Mean fuel burning rate can also be expressed as,

$$\bar{\dot{m}}_F'' = MW_F A_f C_F^{n_1} C_{O_2}^{n_2} e^{-E/R_u T} \quad \dots (19)$$

Expression for burning velocity becomes,

$$S_L = \left[\frac{32\alpha}{9\rho_u} (v+1) \bar{\dot{m}}_F''' \right]^{0.5}$$

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Flame Thickness

Flame Thickness
(δ_L)

Ratio of maximum temperature difference to the temperature difference at the inflection point

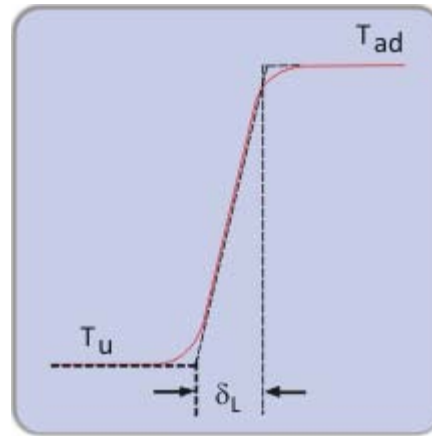
$$\delta_L \equiv \frac{(T_F - T_u)}{(dT/dx)_{ig}}$$

Ignition temperature can be approximated as

$$T_{ig} = 0.75 T_F + 0.25 T_u$$

The temperature gradient at the flame surface is

$$\left. \frac{dT}{dx} \right|_{ig} = \frac{3}{4} \frac{\dot{m}'' C_p}{k_g} (T_F - T_u)$$



(Figure 23.2)

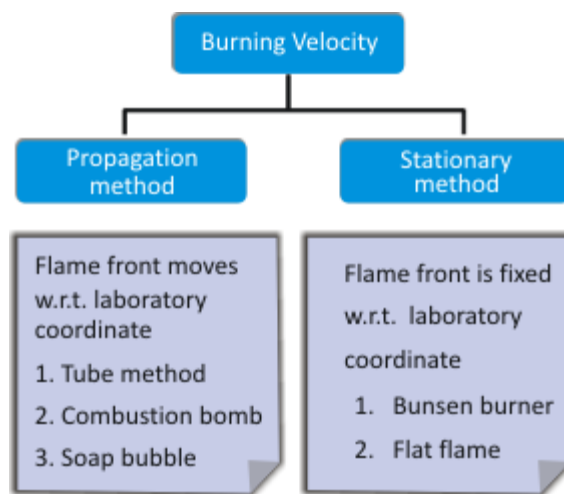
$$\delta_L \equiv \frac{4 k_g}{3 C_p \dot{m}''}$$

$$\delta_L = \frac{4}{3} \frac{\alpha}{S_L}$$

Where, $\alpha = k_g / \rho C_p$

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Burning Velocity Measurement Methods



(Figure 23.3)

Flame front visualization

- Luminous photography
- Shadowgraph photography
- Schlieren Photography

Luminous	<ul style="list-style-type: none"> • Luminous part occurs at the burnt side • Flame speed w.r.t. unburnt gas is needed
Shadowgraph	<ul style="list-style-type: none"> • Corresponds to 2nd derivative of density • Closer to inflection point in temp. profile
Schlieren	<ul style="list-style-type: none"> • Captures maximum density gradient • Closer to unburnt mixture -<i>preferred one</i>